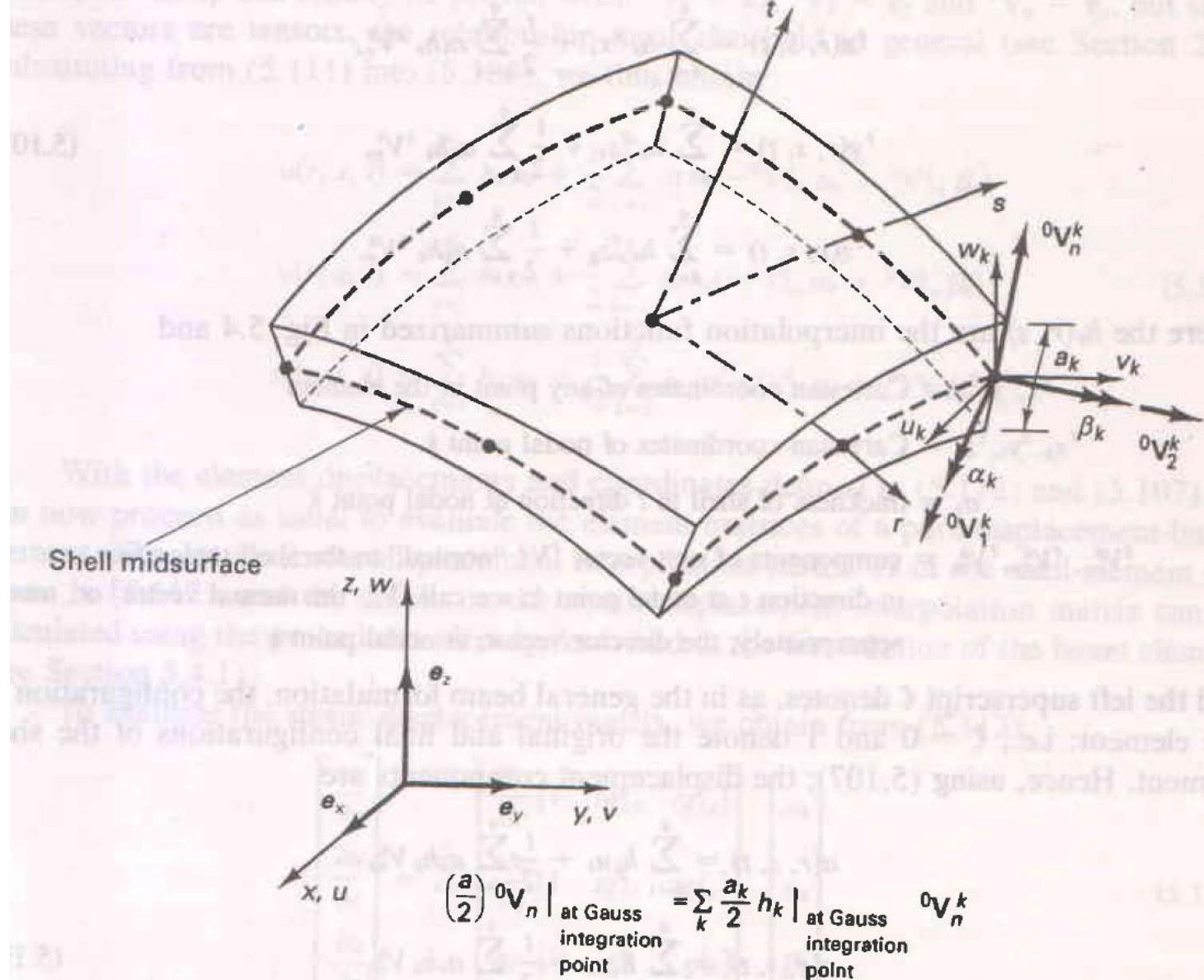


# Shells



**Figure 5.33** Nine-node shell element; also, definition of orthogonal  $\bar{r}, \bar{s}, \bar{t}$  axes for constitutive relations

**EXAMPLE 5.32:** Consider the four-node shell element shown in Fig. E5.32.

- Develop the entries in the displacement interpolation matrix.
- Calculate the thickness at the midpoint of the element and give the direction in which this thickness is measured.

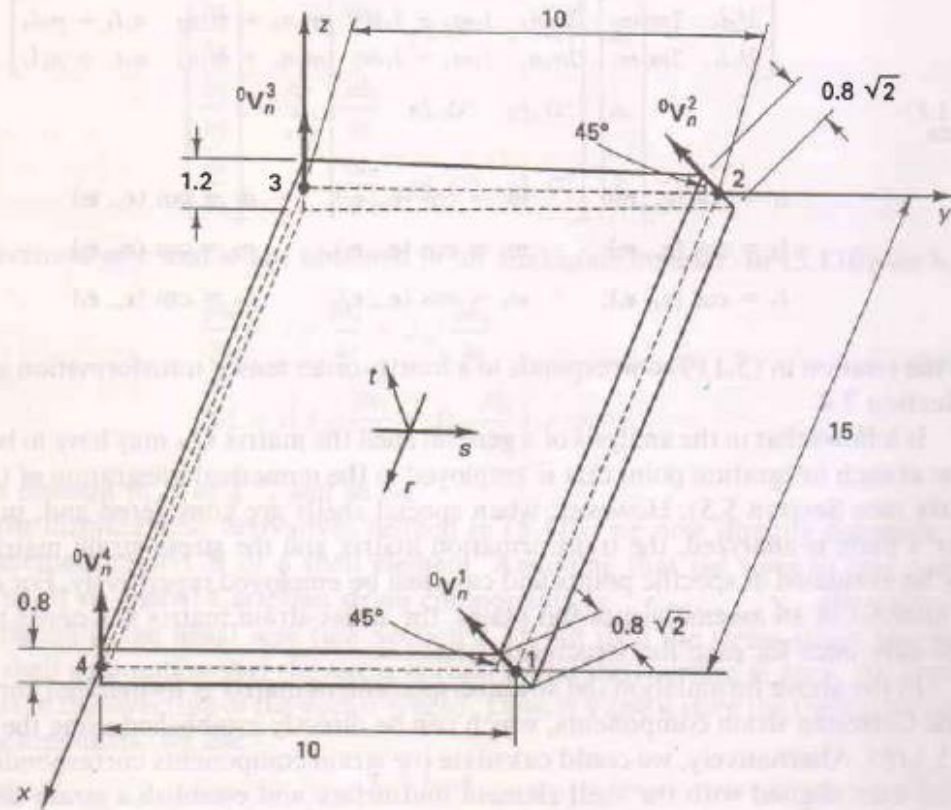


Figure E5.32 Four-node shell element

The shell element considered has varying thickness

$${}^0\mathbf{V}_n^1 = \begin{bmatrix} 0 \\ -1/\sqrt{2} \\ 1/\sqrt{2} \end{bmatrix}; \quad {}^0\mathbf{V}_n^2 = \begin{bmatrix} 0 \\ -1/\sqrt{2} \\ 1/\sqrt{2} \end{bmatrix}; \quad {}^0\mathbf{V}_n^3 = \begin{bmatrix} 0 \\ 0 \\ 1 \end{bmatrix}; \quad {}^0\mathbf{V}_n^4 = \begin{bmatrix} 0 \\ 0 \\ 1 \end{bmatrix}$$

$${}^0\mathbf{V}_1^1 = {}^0\mathbf{V}_1^2 = {}^0\mathbf{V}_1^3 = {}^0\mathbf{V}_1^4 = \begin{bmatrix} 1 \\ 0 \\ 0 \end{bmatrix}$$

$${}^0\mathbf{V}_2^1 = {}^0\mathbf{V}_2^2 = \begin{bmatrix} 0 \\ 1/\sqrt{2} \\ 1/\sqrt{2} \end{bmatrix}; \quad {}^0\mathbf{V}_2^3 = {}^0\mathbf{V}_2^4 = \begin{bmatrix} 0 \\ 1 \\ 0 \end{bmatrix}$$

Also,  $a_1 = a_2 = 0.8\sqrt{2}; \quad a_3 = 1.2; \quad a_4 = 0.8$

The above expressions give all entries in (5.112).

To evaluate the thickness at the element midpoint and the direction in which the thickness is measured, we use the relation

$$\left(\frac{a}{2}\right){}^0\mathbf{V}_n \Big|_{\text{midpoint}} = \sum_{k=1}^4 \frac{a_k}{2} h_k \Big|_{r=s=0} {}^0\mathbf{V}_n^k$$

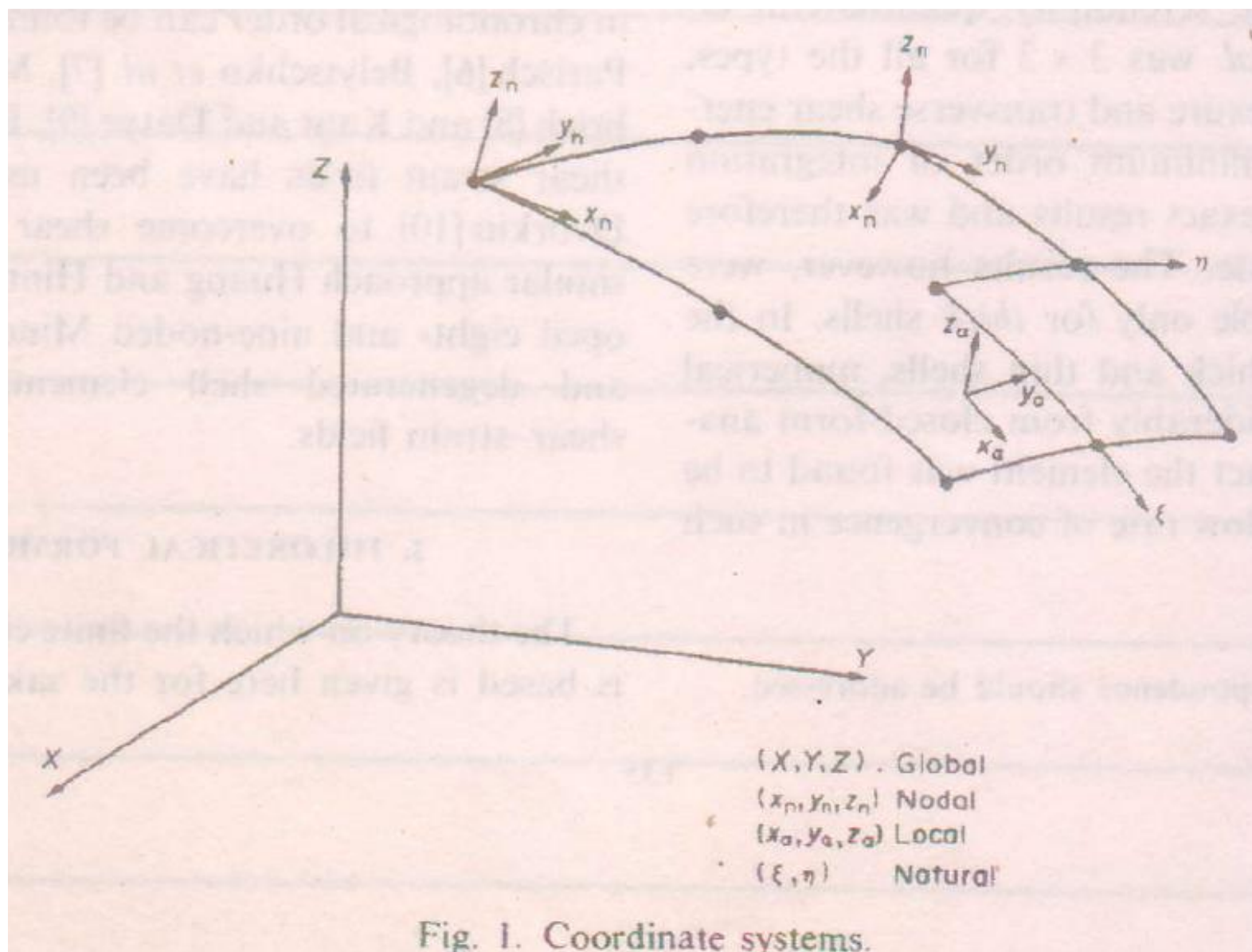
where  $a$  is the thickness and the director vector  ${}^0\mathbf{V}_n$  gives the direction sought. This expression gives

$$\frac{a}{2} {}^0\mathbf{V}_n = \frac{0.8\sqrt{2}}{4} \begin{bmatrix} 0 \\ -1/\sqrt{2} \\ 1/\sqrt{2} \end{bmatrix} + \frac{1.2}{8} \begin{bmatrix} 0 \\ 0 \\ 1 \end{bmatrix} + \frac{0.8}{8} \begin{bmatrix} 0 \\ 0 \\ 1 \end{bmatrix} = \begin{bmatrix} 0 \\ -0.2 \\ 0.45 \end{bmatrix}$$

which gives

$${}^0\mathbf{V}_n = \begin{bmatrix} 0.0 \\ -0.406 \\ 0.914 \end{bmatrix}; \quad a = 0.985$$





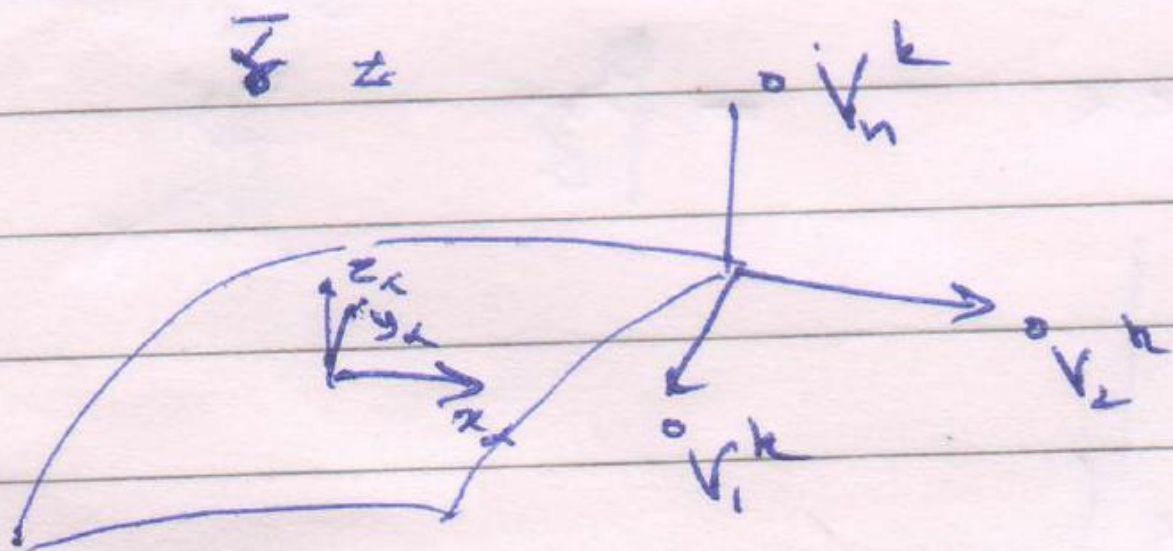
$\vec{r}, \vec{s}, t$  at any point.

A set of tangent vectors to midsurface  
are

$$\vec{\nabla}_r = \frac{\partial x}{\partial r} \vec{i} + \frac{\partial y}{\partial r} \vec{j} + \frac{\partial z}{\partial r} \vec{k}$$

tangent to  $r$

$$\vec{\nabla}_{\vec{r}, \vec{s}} = \left( \sum \frac{\partial h_k}{\partial x} x_k \right) \vec{i} + \left( \sum \frac{\partial h_k}{\partial y} y_k \right) \vec{j} + \left( \sum \frac{\partial h_k}{\partial z} z_k \right) \vec{k}$$



$\vec{k}_\alpha$  is along  $\vec{v}_n^k$  at that point  
 $i_\alpha$  is along  $\vec{v}_1^k$   
 $J_\alpha$  is along  $\vec{k}_\alpha \times \vec{i}_\alpha$



$$u_\alpha(x_\alpha, y_\alpha, z_\alpha) = u_{0\alpha}(x_\alpha, y_\alpha) - z_\alpha \beta_{yx}(x_\alpha, y_\alpha)$$

$$= \sum_k h_k u_{0\alpha}^k + z_\alpha \sum_k h_k \phi_y^k$$

$$v_\alpha(x_\alpha, y_\alpha, z_\alpha) = v_{0\alpha}(x_\alpha, y_\alpha) - z_\alpha \beta_{yx}(x_\alpha, y_\alpha)$$

$$= \sum_k h_k v_{0\alpha}^k - z_\alpha \sum_k h_k \phi_x^k$$

$$w_\alpha(x_\alpha, y_\alpha, z_\alpha) = w_{0\alpha}(x_\alpha, y_\alpha)$$

$$\epsilon_{2\alpha} = \epsilon_{20\alpha} - z_{\alpha} \frac{\partial \beta_{2\alpha}}{\partial x_{\alpha}}$$

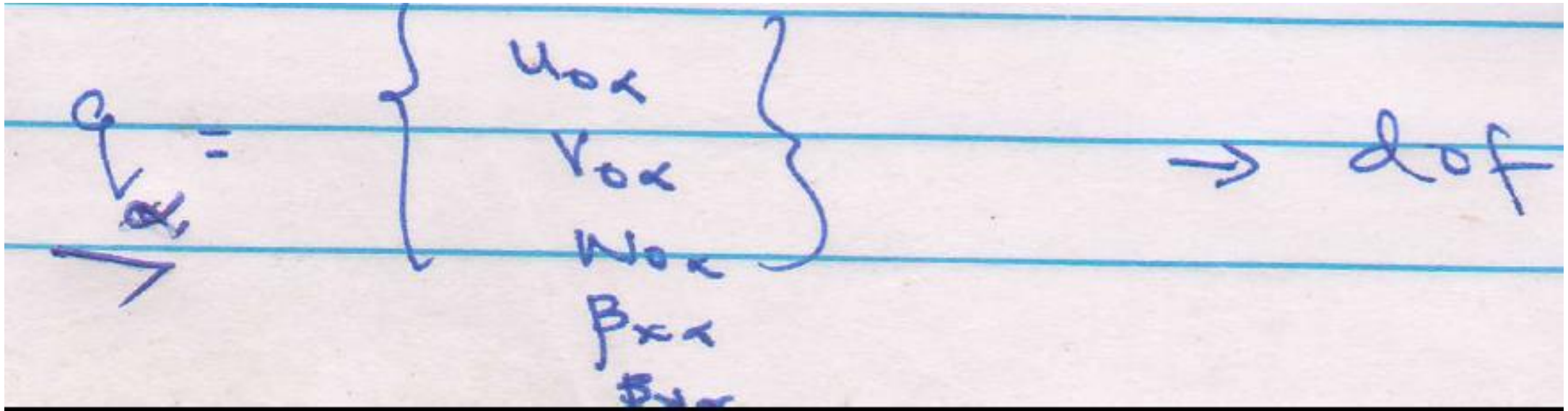
$$\epsilon_{y\alpha} = \epsilon_{y0\alpha} - z_{\alpha} \beta_{y\alpha, y\alpha}$$

$$\delta_{xy\alpha} = \delta_{xy0\alpha} - z_{\alpha} (\beta_{x\alpha, y\alpha} + \beta_{y\alpha, x\alpha})$$

$$\delta_{xz\alpha} = -\beta_{x\alpha} + \frac{\partial \omega_{\alpha}}{\partial x_{\alpha}}$$

$$\delta_{yz\alpha} = -\beta_{y\alpha} + \frac{\partial \omega_{\alpha}}{\partial y_{\alpha}}$$

- At any point on shell surface there are 5 dof





Int v.w

$$= \int \left[ \delta \epsilon_{xx} \sigma_{xx} + \delta \epsilon_{yy} \sigma_{yy} + \delta \gamma_{xy} \tau_{xy} + \delta \gamma_{xz} \tau_{xz} + \delta \gamma_{yz} \tau_{yz} \right] dV$$

$$= \iint \left\{ \sigma_{xx} (\delta \epsilon_{xx} - z \rho_{xx}) + \sigma_{yy} (\delta \epsilon_{yy} - z \rho_{yy}) + \tau_{xy} [\delta \gamma_{xy} - z (\rho_{yx} + \rho_{xy})] + \tau_{yz} [\delta \gamma_{yz}] + \tau_{xz} \delta \gamma_{xz} \right\} dz dA$$



$$\begin{aligned}
&= \int_A \delta E_{x\alpha} N_{x\alpha} + \delta E_{y\alpha} N_{y\alpha} + \delta E_{xy\alpha} N_{xy\alpha} \\
&\quad + M_{x\alpha} (-\delta \beta_{x\alpha, x\alpha}) + M_{y\alpha} (-\delta \beta_{y\alpha, y\alpha}) \\
&\quad + M_{xy\alpha} [-(\delta \beta_{y\alpha, x\alpha} + \delta \beta_{x\alpha, y\alpha})] \\
&\quad + \left. \begin{aligned} &\delta_{xz} Q_{z\alpha} + \delta_{yz} Q_{y\alpha} \end{aligned} \right\} \delta A
\end{aligned}$$

$$N_{xx} = \int_{-t_x/2}^{t_x/2} \sigma_{xx} dz \quad \text{etc.}$$

$$\begin{aligned}
 N_\alpha &= \begin{Bmatrix} N_{xx} \\ N_{yx} \\ N_{xy\alpha} \end{Bmatrix} = \begin{bmatrix} [D_{m\alpha}]_{3 \times 3} & \begin{matrix} \text{circle} \\ 3 \times 3 \end{matrix} & \begin{matrix} \text{circle} \\ 3 \times 3 \end{matrix} \end{bmatrix} \begin{Bmatrix} \epsilon_{0xx} \\ \epsilon_{0yx} \\ \epsilon_{0xy,\alpha} \end{Bmatrix} \\
 M_\alpha &= \begin{Bmatrix} M_{xx} \\ M_{yx} \\ M_{xy\alpha} \end{Bmatrix} = \begin{bmatrix} \begin{matrix} \text{circle} \\ 3 \times 3 \end{matrix} & [D_{b\alpha}]_{3 \times 3} & \begin{matrix} \text{circle} \\ 3 \times 3 \end{matrix} \end{bmatrix} \begin{Bmatrix} \beta_{x,x} \\ \beta_{y,y} \\ \beta_{xy} + \beta_{yx} \end{Bmatrix} \\
 Q_\alpha &= \begin{Bmatrix} Q_{xx} \end{Bmatrix} = \begin{bmatrix} [D_{s\alpha}]_{3 \times 3} & [0]_{3 \times 1} & [0]_{3 \times 1} \end{bmatrix} \begin{Bmatrix} \gamma_{xz} \\ \delta_{yz} \end{Bmatrix}
 \end{aligned}$$

$$\mathbf{D}_{mz} = \begin{bmatrix} \frac{Et_z}{(1-\nu\nu)} & \frac{\nu Et_z}{(1-\nu\nu)} & 0 & 0 & 0 & 0 & 0 & 0 \\ \frac{\nu Et_z}{(1-\nu\nu)} & \frac{Et_z}{(1-\nu\nu)} & 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & \frac{Et_z}{2(1+\nu)} & 0 & 0 & 0 & 0 & 0 \end{bmatrix}$$

$$\mathbf{D}_{lx} = \begin{bmatrix} 0 & 0 & 0 & \frac{Et_x^3}{12(1-\nu\nu)} & \frac{\nu Et_x^3}{12(1-\nu\nu)} & 0 & 0 & 0 \\ 0 & 0 & 0 & \frac{\nu Et_x^3}{12(1-\nu\nu)} & \frac{Et_x^3}{12(1-\nu\nu)} & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & \frac{Et_x^3}{24(1+\nu)} & 0 & 0 \end{bmatrix}$$

$$\mathbf{D}_{yz} = \begin{bmatrix} 0 & 0 & 0 & 0 & 0 & 0 & \frac{Et_z}{2.4(1+\nu)} & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & 0 & \frac{Et_z}{2.4(1+\nu)} \end{bmatrix}$$



$$\begin{aligned}\bar{E}_{m\alpha} &= B_{m\alpha} \bar{q}_\alpha \\ \bar{E}_{b\alpha} &= B_{b\alpha} q_\alpha \\ \bar{r}_{s\alpha} &= B_{s\alpha} q_\alpha\end{aligned}$$

$$q_\alpha = \begin{Bmatrix} u_{0\alpha} \\ v_{0\alpha} \\ w_{0\alpha} \\ \bar{p}_{\alpha} \\ \bar{p}_{\alpha} \end{Bmatrix}$$

$$\begin{Bmatrix} u_{0\alpha} \\ v_{0\alpha} \\ w_{0\alpha} \\ \bar{p}_{y\alpha} \\ \bar{p}_{x\alpha} \end{Bmatrix} = \sum_{\alpha=1}^N \begin{bmatrix} h_i(r,s) u_{i\alpha} \\ h_i(r,s) v_{i\alpha} \\ h_i(r,s) w_{i\alpha} \\ -h_i(r,s) \bar{p}_{x\alpha} \\ h_i(r,s) \bar{p}_{y,\alpha} \end{bmatrix}$$



$$= \sum_{i=1}^2 \begin{bmatrix} h_i & 0 & 0 & 0 & 0 \\ 0 & h_i & 0 & 0 & 0 \\ 0 & 0 & h_i & 0 & 0 \\ 0 & 0 & 0 & -h_i & 0 \\ 0 & 0 & 0 & 0 & h_i \end{bmatrix} \begin{Bmatrix} u_{ix} \\ v_{ix} \\ w_{ix} \\ \theta_{ix} \\ \phi_{ix} \end{Bmatrix}$$

$$\underline{v}_\alpha = \sum_{i=1}^2 h_i \underline{v}_{ix}$$

Let displacement at node 'i' in nodal  
 coord system

$$q_{in} = [u_{on} \quad v_{on} \quad w_{on} \quad \theta_{xn} \quad \theta_{yn}]$$

$$q_{ia} = R_{ai} q_{in}$$

$$q_a = \sum_{i=1}^N k_i R_{ai} q_{in}$$

$$R_n = \begin{bmatrix} \mathbf{i}_1 \cdot \mathbf{i}_n & \mathbf{i}_1 \cdot \mathbf{j}_n & \mathbf{i}_1 \cdot \mathbf{k}_n & 0 & 0 \\ \mathbf{j}_1 \cdot \mathbf{i}_n & \mathbf{j}_1 \cdot \mathbf{j}_n & \mathbf{j}_1 \cdot \mathbf{k}_n & 0 & 0 \\ \mathbf{k}_1 \cdot \mathbf{i}_n & \mathbf{k}_1 \cdot \mathbf{j}_n & \mathbf{k}_1 \cdot \mathbf{k}_n & 0 & 0 \\ 0 & 0 & 0 & \mathbf{i}_1 \cdot \mathbf{i}_n & \mathbf{i}_1 \cdot \mathbf{j}_n \\ 0 & 0 & 0 & \mathbf{j}_1 \cdot \mathbf{i}_n & \mathbf{j}_1 \cdot \mathbf{j}_n \end{bmatrix}$$

$$\underline{E}_m = \underline{L}_m \underline{q} = \underline{L}_m \sum_{i=1}^N h_i \underline{q}_i$$

$$= \sum_{i=1}^N \underline{B}_{mi} \underline{q}_i$$

$$= \sum_{i=1}^N \underline{B}_{mi} R_i \underline{q}_i$$

$$= \sum_{i=1}^N \underline{B}_{mi}^* \underline{q}_i = [\underline{B}_m^*] \{ \underline{q}_i \}$$



Similarly,

$$\begin{aligned} E_{b\alpha} &= [B_b^*] \{q_n\} \\ E_{s\alpha} &= [B_s^*] \{q_n\} \end{aligned}$$

$$\begin{aligned} \bar{E}_\alpha &= [B^*] \{q_n\} \\ &= [B_m^* \quad B_b^* \quad B_s^*]^T \{q_n\} \end{aligned}$$



$$\mathbf{L}_{m1} = \begin{bmatrix} \frac{\partial}{\partial x_1} & 0 & 0 & 0 & 0 \\ 0 & \frac{\partial}{\partial y_1} & 0 & 0 & 0 \\ \frac{\partial}{\partial y_1} & \frac{\partial}{\partial x_1} & 0 & 0 & 0 \end{bmatrix}$$

$$\mathbf{L}_{t2} = \begin{bmatrix} 0 & 0 & 0 & 0 & \frac{\partial}{\partial x_1} \\ 0 & 0 & 0 & -\frac{\partial}{\partial y_1} & 0 \\ 0 & 0 & 0 & -\frac{\partial}{\partial x_1} & \frac{\partial}{\partial y_1} \end{bmatrix}$$

$$\mathbf{L}_{y2} = \begin{bmatrix} 0 & 0 & \frac{\partial}{\partial x_1} & 0 & 1 \\ 0 & 0 & \frac{\partial}{\partial y_1} & -1 & 0 \end{bmatrix}$$

You may notice that we have ignored a sixth row in R matrix. Actually, it should also be there and would transform the rotation  $\theta_{x\alpha}$  and  $\theta_{y\alpha}$  along the normal at the node. The normal at a node does not really coincide with normal at  $(x_\alpha, y_\alpha, z_\alpha)$ . However this component of rotation along node normal may be small as angle between the node normal and normal at  $(x_\alpha, y_\alpha, z_\alpha)$  may be small. By rotating to nodal coordinate system continuity of variables  $(u_n, v_n, w_n, \alpha, \beta)$  is maintained.

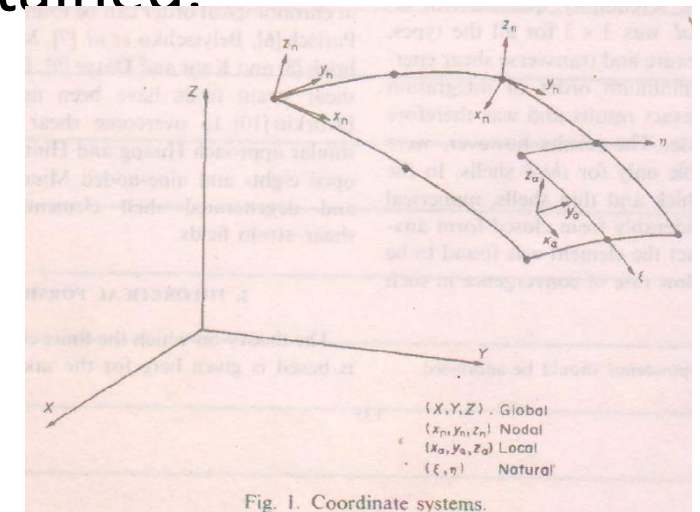
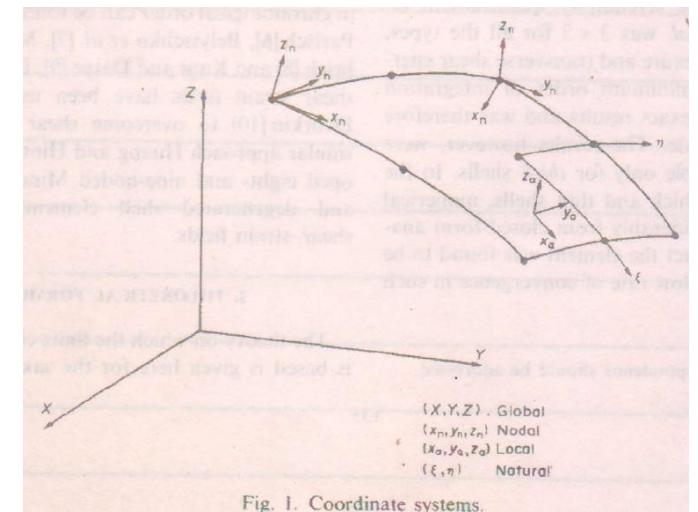


Fig. 1. Coordinate systems.

- Instead of rotating to nodal coordinate system we can also rotate to global coordinate system and we have six dofs.





$$\mathbf{K}^e = \int_A \mathbf{B}^{*T} \mathbf{D}_2^e \mathbf{B}^* dA$$

$$\mathbf{K}^e = \mathbf{K}_m^e + \mathbf{K}_b^e + \mathbf{K}_s^e,$$

$$\mathbf{K}_m^e = \int_{-1}^{+1} \int_{-1}^{+1} \mathbf{B}_m^{*T} \mathbf{D}_{m2} \mathbf{B}_m^* |\mathbf{J}| d\xi d\eta$$

$$\mathbf{K}_m^e = \sum_{\beta=1}^m w_{\beta} (\mathbf{B}_m^*)'_{\beta} (\mathbf{D}_{m2})_{\beta} (\mathbf{B}_m^*)_{\beta} |\mathbf{J}|_{\beta},$$

$$\mathbf{K}_b^e = \sum_{\beta=1}^m w_{\beta} (\mathbf{B}_b^*)'_{\beta} (\mathbf{D}_{b2})_{\beta} (\mathbf{B}_b^*)_{\beta} |\mathbf{J}|_{\beta}$$

$$\mathbf{K}_s^e = \sum_{\beta=1}^m w_{\beta} (\mathbf{B}_s^*)'_{\beta} (\mathbf{D}_{s2})_{\beta} (\mathbf{B}_s^*)_{\beta} |\mathbf{J}|_{\beta}.$$

Here  $\mathbf{K}_m^e$ ,  $\mathbf{K}_b^e$ ,  $\mathbf{K}_s^e$  are the element membrane, bending and shear stiffness matrices, respectively,  $m$  is the number of Gauss points and  $w_{\beta}$  are the weights of Gauss points. Thus, any order of integration (either

$2 \times 2$  or  $3 \times 3$ ) can be independently applied to calculate any of the individual  $\mathbf{K}$  matrix.

Load vector

Body forces

$$\int [u \quad v \quad w] \begin{Bmatrix} -\rho g_{x\alpha} \\ -\rho g_{y\alpha} \\ -\rho g_{z\alpha} \end{Bmatrix} dx_\alpha dy_\alpha dz_\alpha$$

$$= \int [u_{0x} \ v_{0x} \ w_{0x}] \begin{bmatrix} -\rho g_{xx} \\ -\rho g_{yx} \\ -\rho g_{zx} \end{bmatrix} dx_x dy_x dz_x$$

$$+ \int \int_{-H_z}^{t/2} \begin{bmatrix} -z \rho_x & -z \rho_y & 0 \end{bmatrix} \begin{bmatrix} \end{bmatrix} dz_x$$

0  $(\because \frac{z^2}{2} \Big|_{-H_z}^{t/2} = 0)$

$$= \int [u_{0x} \ v_{0x} \ w_{0x}] \begin{bmatrix} -\rho g_{xx} \\ -\rho g_{yx} \\ -\rho g_{zx} \end{bmatrix} dx_x dy_x$$

$$= \int \begin{bmatrix} u_{0x} \\ v_{0x} \\ w_{0x} \end{bmatrix} \begin{bmatrix} h_1 & 0 & 0 \\ 0 & h_1 & 0 \\ 0 & 0 & h_1 \\ 0 & 0 & 0 \\ 0 & 0 & 0 \\ h_2 & 0 & 0 \\ 0 & h_2 & 0 \\ 0 & 0 & h_2 \end{bmatrix} \begin{bmatrix} -\rho g_{xx} \\ -\rho g_{yx} \\ -\rho g_{zx} \end{bmatrix} dx_x dy_x$$



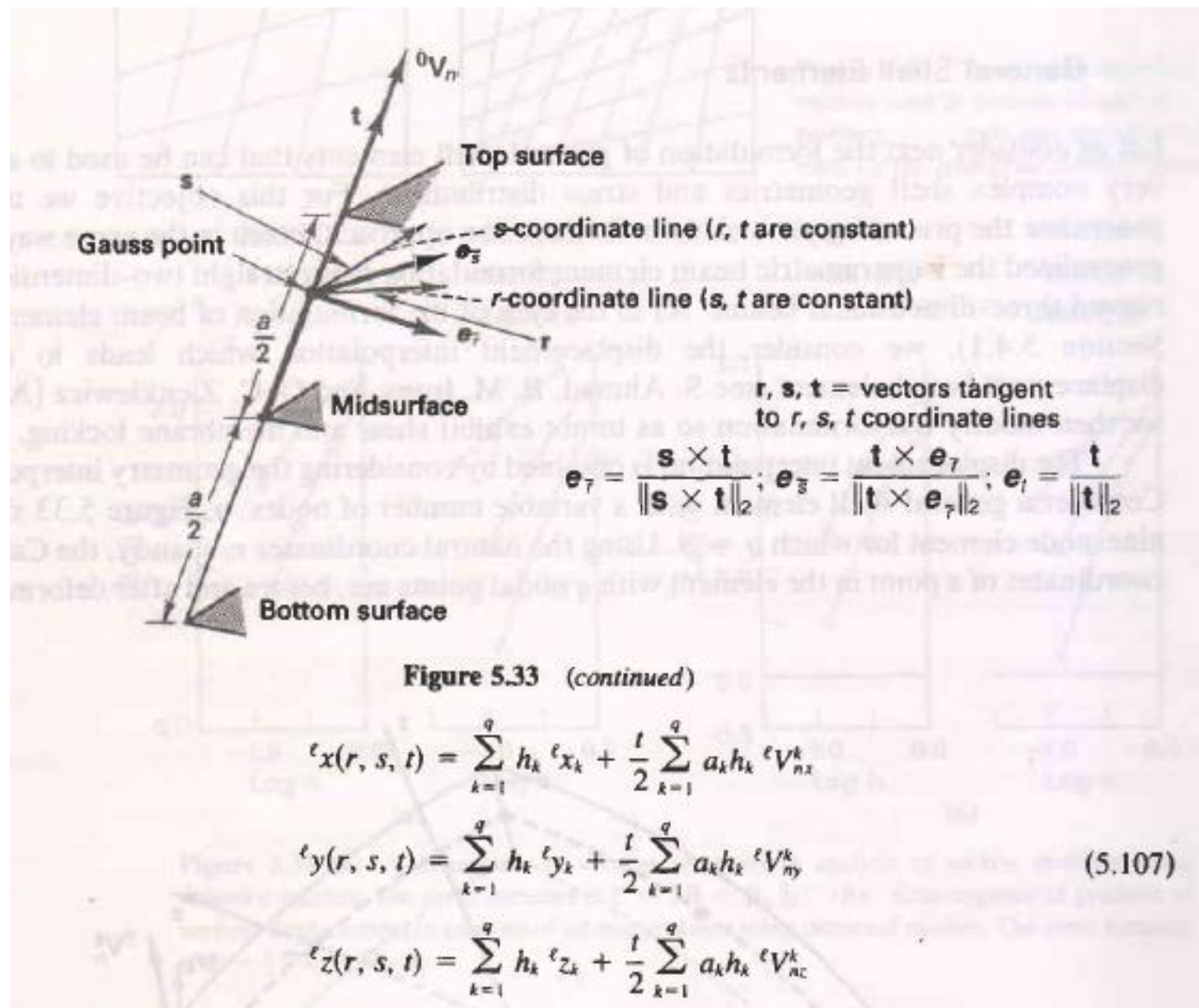


$$|J| = \begin{vmatrix} l_{1x} \frac{\partial x}{\partial r} + m_{1x} \frac{\partial y}{\partial r} + n_{1x} \frac{\partial z}{\partial r} & l_{2x} \frac{\partial x}{\partial r} + m_{2x} \frac{\partial y}{\partial r} + n_{2x} \frac{\partial z}{\partial r} \\ l_{1x} \frac{\partial x}{\partial s} + m_{1x} \frac{\partial y}{\partial s} + n_{1x} \frac{\partial z}{\partial s} & l_{2x} \frac{\partial x}{\partial s} + m_{2x} \frac{\partial y}{\partial s} + n_{2x} \frac{\partial z}{\partial s} \end{vmatrix}$$

- The coordinates x, y in terms of nodal coordinates are given below.  
On the mid-plane put  $t=0$  in the expressions below.

- Loads at the boundary due to moments or transverse loads can be resolved into moments or forces in x, y, z directions.





${}^e x, {}^e y, {}^e z$  = Cartesian coordinates of any point in the element

${}^e x_k, {}^e y_k, {}^e z_k$  = Cartesian coordinates of nodal point  $k$

$a_k$  = thickness of shell in  $t$  direction at nodal point  $k$

${}^e V_{nx}^k, {}^e V_{ny}^k, {}^e V_{nz}^k$  = components of unit vector  ${}^e \mathbf{V}_n^k$  "normal" to the shell midsurface in direction  $t$  at nodal point  $k$ ; we call  ${}^e \mathbf{V}_n^k$  the normal vector' or, more appropriately, the director vector, at nodal point  $k$

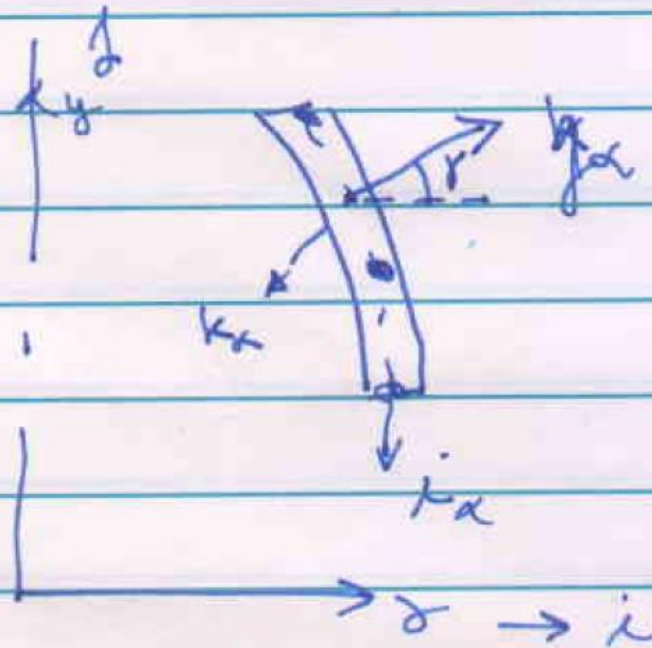
$\ell = 0$  and  $1$  denote the original and final configurations of the shell

$$\begin{aligned}
 u(r, s, t) &= \sum_{k=1}^q h_k u_k + \frac{t}{2} \sum_{k=1}^q a_k h_k V_{nx}^k \\
 v(r, s, t) &= \sum_{k=1}^q h_k v_k + \frac{t}{2} \sum_{k=1}^q a_k h_k V_{ny}^k \\
 w(r, s, t) &= \sum_{k=1}^q h_k w_k + \frac{t}{2} \sum_{k=1}^q a_k h_k V_{nz}^k
 \end{aligned} \tag{5.108}$$



# Axisymmetric shell (HW)

For a 3-node axisymmetric shell element



$$\frac{\partial \alpha}{\partial \theta} = \cos \theta$$

etc.

Let  $\theta_\alpha$  be the rotation of the normal about the  $k_\alpha$  axis

$k_\alpha$  is perpendicular to plane of paper

$$u_\alpha = u_{0\alpha} - z_\alpha \beta_{xx}$$

$$w_\alpha = w_{0\alpha}(x_\alpha)$$

$$\left\{ \theta_\alpha \right\} = \left\{ \begin{matrix} u_{0\alpha} \\ w_{0\alpha} \\ \beta_{xx} \end{matrix} \right\}$$

Following the procedure followed for shell element, determine the membrane and bending strain in r-y coordinate system. Hence determine the stiffness matrix.

Beam K.E and Mass Matrix

$$= \frac{1}{2} \int \rho (\dot{u}^2 + \dot{w}^2) dV$$

$$= \frac{1}{2} \int \rho (\dot{u}^T \dot{u} + \dot{w}^T \dot{w}) b dz dx$$

$$= \frac{1}{2} \int \int_{-h/2}^{h/2} \rho \left( -z \frac{d\dot{\beta}}{dx} \right)^T \left( -z \frac{d\dot{\beta}}{dx} \right) b dz dx$$

$$+ \frac{1}{2} \int \int_{-h/2}^{h/2} \rho \dot{w}^T \dot{w} b dz dx$$



$$= \frac{1}{2} \iiint \rho z^2 [\dot{w}_1 \dot{w}_2 \dot{\beta}_1 \dot{\beta}_2] H_b^T H_b \begin{bmatrix} \dot{z}_1 \\ \dot{z}_2 \\ \dot{\beta}_1 \\ \dot{\beta}_2 \end{bmatrix} b \, dz \, dx$$

$$+ \frac{1}{2} \int \rho \left[ \right] H_w^T H_w \left[ \right] b \, dz \, dx$$

$$M = \frac{1}{2} \int \frac{\rho b h^3}{12} \left[ \right] H_b^T H_b \left[ \right] dx + \frac{1}{2} \int \rho h \left[ \right] H_w^T H_w \left[ \right] dx$$

$$M = \int \begin{bmatrix} H_w \\ H_b \end{bmatrix}^T \begin{bmatrix} \rho b h & \\ & \frac{\rho b h^3}{12} \end{bmatrix} \begin{bmatrix} H_w \\ H_b \end{bmatrix} |det J| \, dr$$

HW

Using sumlaw approach for shell  
show that

$$E = \frac{1}{2} \int \underline{\dot{u}}^T \rho \underline{\dot{u}} \, dV$$

$$= \frac{1}{2} \int \dot{q}_x^T M \dot{q}_x \, dA$$

$$m = \begin{bmatrix} I_1 & 0 & 0 & 0 & I_2 \\ 0 & I_1 & 0 & -I_2 & 0 \\ 0 & 0 & I_1 & 0 & 0 \\ 0 & -I_2 & 0 & I_3 & 0 \\ I_2 & 0 & 0 & 0 & I_3 \end{bmatrix}$$

$$I_1 = \int \rho \, dz; \quad I_2 = \int \rho z \, dz; \quad I_3 = \int \rho z * z \, dz.$$